# **WORKSHOP PROBLEM 4a**

# Linear Buckling Load Analysis (without spring)



**Objectives:** 

- Create and prepare the appropriate model for the analysis.
- Demonstrate the use of linear buckling analysis.

**4a-2** MSC/NASTRAN for Windows 103 Exercise Workbook

# **Model Description:**

Below in Figure 4a.1 is a finite element representation of a structure composed of a cantilever beam and a spring. A load will be applied at the junction of the beam and the spring. In this exercise, a linear buckling analysis will be performed on the model without the spring element.





Table 4a.1 - Properties

Elastic Modulus:	10.E7 psi
Bar Cross Sectional Area:	0.1 in <sup>2</sup>
Load, P:	6 lbs.

# **Exercise Procedure:**

1. Start up MSC/NASTRAN for Windows V3.0 and begin to create a new model.

Double click on the icon labeled MSC/NASTRAN for Windows V3.0.

On the Open Model File form, select New Model.

**Open Model File:** 

New Model

2. Create a material called **mat\_1**.

From the pulldown menu, select Model/Material.

# Model/Material...

Title:

Youngs Modulus:

mat_1	
10.E7	

OK	
Cancel	

3. Create the property that will define the beam element.

# Model/Property...

Elem/Property Type		
Line Elements:	• Rod	
ОК		
Title:	prop_1	

To select the material, click on the list icon next to the databox and select **mat\_1**.

Material:

Area:

OK Cancel

1mat_1	
0.1	

4. Create the NASTRAN finite element model.

#### Mesh/Between...

Property:	1prop_	1	
Mesh Size/#Nodes/Dir1:	2		
ОК			
	<i>X</i> :	<i>Y</i> :	<i>Z</i> :
Corner 1:	0	0	0
ОК			
	<i>X</i> :	<i>Y</i> :	<i>Z</i> :
Corner 2:	100	1	0
ОК			

To bring the model into the viewable area, use the Autoscale feature.

#### View/Autoscale

5. Create the model constraints.

Before creating the appropriate constraints, a constraint set must be created by performing the following:

# Model/Constraint/Set...

Title:

constraint\_1

OK

Now define the relevant constraint for the model.

#### Model/Constraint/Nodal...

Select Node 1.

ОК	
	Fixed
OK	

Next, select Node 2.

ОК	
DOF:	🔀 TX 🗌 TY 🔀 TZ
	🛛 RX 🕅 RY 🕅 RZ
OK	

6. Create the model loading.

Like the constraints, a load set must first be generated before creating the appropriate model loading.

#### Model/Load/Set...

Title:

Cancel

load\_1

OK

Next, create the load.

# Model/Load/Nodal...

Select Node 2.

OK

Highlight Force.



OK

-6

Cancel

7. Submit the job for analysis.

#### File/Export/Analysis Model...

Analysis Type:



OK

Change the directory to C:\temp.

File name:	prob4a
Write	
	🔀 Run Analysis
Advanced	
Define the buckling analy	sis parameters.

Method ID:

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Model Solution Method:

1	
• Inverse Power	

Under Range of Interest, enter the following:

From:

To:

0	
3	

Now enter the number of eigenvalues and eigenvectors to be estimated under *Eigenvalues and Eigenvectors*.

Number Estimated:

Number Desired:

20 2

OK

Problem ID:

OK	

Linear Buckling Load Analysis w/o Spring

Under *Output Requests*, change the output to:

2..Print and PostProcess

Also deselect all the boxes except the following:



OK

OK

When asked if you wish to save the model, respond Yes.

File name:

prob4a

Save

When the MSC/NASTRAN manager is through running, MSC/ NASTRAN will be restored on your screen, and the *Message Review* form will appear. To read the messages, you could select **Show Details**. Since the analysis ran smoothly, we will not bother with the details this time.

# Continue

When asked if it is "OK to Begin Reading File C:\TEMP\prob4a.xdb," respond **Yes**.

# Yes

8. List the results of the analysis.

To list the results, select the following:

#### List/Output/Query...

Output Set:	1MSC/NASTRAN Case 1
Category:	1Displacement
Entity:	• Node
ID:	2
ОК	

**NOTE:** You may want to expand the message box in order to view the results. To do this, double click on the message box. Adjust the size of the box to your preference by dragging the top border downward.

Answer the following questions using the results. The answers are listed at the end of the exercise.

What is the T2 displacement **Node 2**?

T2 displacement @ Node 2 = \_\_\_\_\_

You can use the query function to find the answer to the remaining questions.

# List/Output/Query...

*Output Set:* 

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2..Eigenvalue

What is the first eigenvalue obtained from the analysis?

EIG = \_\_\_\_\_

What is the critical buckling load (eigenvalue \* applied load)?

P<sub>cr</sub> = \_\_\_\_\_

Click **Cancel** when you are done.

Cancel

9. Plot the deformation of the beam.

View/Options...

Quick Options... Labels Off

Deselect the following:

	Load - Force
$\square$	Constraint



Create the plot.

# View/Select...



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Deformed and Contour Data	
Data Selection/Category:	1Displacement
Output Set:	1MSC/NASTRAN Case 1
Output Vectors/Deformation:	3T2 Translation
Output Vectors/Contour:	3T2 Translation
OK	

In order to see the deformation results accurately, you will need to turn off the display scaling of the actual deformation.

# View/Options...

Category:

Options:

• PostProcessing

**Deformed Style** 

% of Model (Actual)

OK

OK

The XY View should appear as follows:

# Figure 4a.2



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As seen from the fringe values, the beam has a maximum downward deflection of 0.600. Since the load is less than the calculated critical load, the beam does not "snap-through" the maximum compression (deflection = 1).

This concludes the exercise.

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Disp Y @ Node 2:	-0.60009
Eigenvalue:	1.66658
Critical Load:	9.99948